
Finite Element Methods: Numerical Exercises

Paul J. Atzberger

1. Show that each of the elements have the stated regularity as follows:

- (a) Lagrange triangular element based on \mathcal{P}_k with $k + 1$ distinct nodes along each edge is C^0 .
- (b) Hermite triangular element based on \mathcal{P}_3 is C^0 .
- (c) Argyris triangular element based on \mathcal{P}_5 is C^1 in the normal direction across edges.

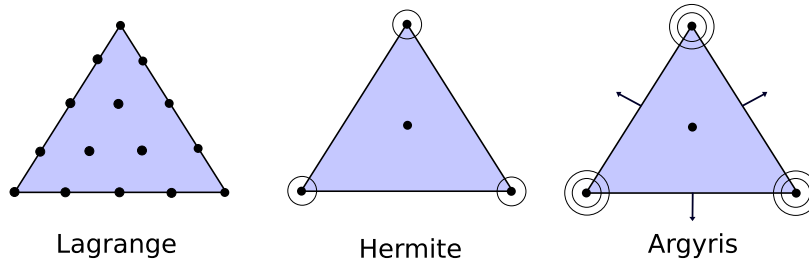


Figure 1: Triangular Elements.

2. There are many ways to develop quadratures for triangulations \mathcal{T} to approximate

$$\int \int_{\mathcal{T}_0} f(\mathbf{x}) d\mathbf{x} \approx \sum_k w_k f(\mathbf{x}_k), \quad \mathbf{x} = (x_1, x_2).$$

- (a) Consider Duffy's Transform from a reference triangular element to a quadrilateral element as shown in Figure 2. This is given by

$$\begin{aligned} \xi &= \left(\frac{1 + \xi'}{2} \right) \left(\frac{1 - \eta'}{2} \right), \quad \eta = \frac{1 + \eta'}{2} \\ \xi' &= \frac{2\xi}{1 - \eta} - 1, \quad \eta' = 2\eta - 1, \end{aligned}$$

where $\eta \in [0, 1]$, $\xi \in [0, 1 - \eta]$, $\xi', \eta' \in [0, 1]$. We can express integration over the triangular element as

$$\int_0^1 \int_0^{1-\eta} f(\xi, \eta) d\eta d\xi = \int_{-1}^1 \int_{-1}^1 f(\xi, \eta) J(\xi', \eta') d\eta' d\xi',$$

where the Jacobian for Duffy's Transform is given by $J(\xi', \eta') = \frac{1}{8}(1 - \eta')$.

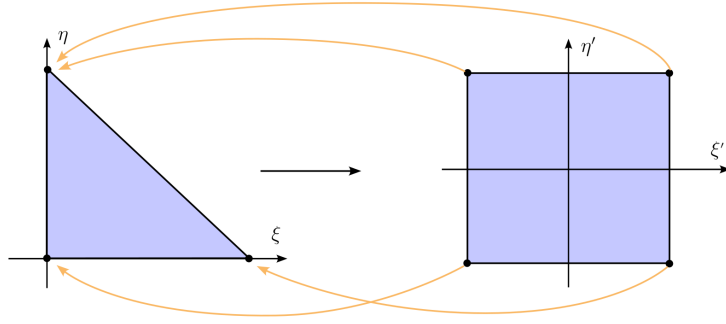


Figure 2: Duffy's Transform.

Use Gaussian quadratures for the cases of 2 and 3 nodes to construct quadratures for the iterated integrals for the quadrilateral. Determine the corresponding nodes and weights for the triangle and construct a quadrature table for the triangular elements for the Gaussian quadrature cases.

- (b) Alternatively, we can use for the weights w_k and nodes \mathbf{x}_k from Table 1. For $n = 4, 7$, compare this with the Duffy's Transform approach for the test functions (i) $3x^3y^2$, (ii) $\sin(\pi xy/2)$, and (iii) $\exp(-3x^2 + 3y^2)$. In each case, which yields the more accurate approximation.

d	n	k	\mathbf{x}_k	ω_k	k	\mathbf{x}_k	w_k	k	\mathbf{x}_k	w_k	k	\mathbf{x}_k	w_k
1	1	1	(1/3,1/3)	1/2									
2	3	1	(1/6,1/6)	1/6	2	(2/3,1/6)	1/6	3	(1/6,2/3)	1/6			
3	4	1	(1/3,1/3)	-9/32	2	(3/5,1/5)	25/96	3	(1/5,3/5)	25/96	4	(1/5,1/5)	25/96
4	7	1	(0,0)	1/40	2	(1/2,0)	1/15	3	(1,0)	1/40	7	(1/3,1/3)	9/40
		4	(1/2,1/2)	1/15	5	(0,1)	1/40	6	(0,1/2)	1/15			

Table 1: Quadratures on triangulations for $\int_0^1 \int_0^{1-x_1} f(\mathbf{x}) d\mathbf{x} \approx \sum_k f(\mathbf{x}_k) w_k$, $\mathbf{x} = (x_1, x_2)$. The d is the quadrature order, n number of nodes, \mathbf{x}_k nodes, and ω_k weights. For affine reference element map $\mathbf{x} = \psi(\mathbf{X})$ with $\psi(\mathcal{T}_\ell) = \mathcal{T}_0$ and Jacobian $J(\mathbf{X}) = |\det \partial\psi/\partial\mathbf{X}|$, the quadrature is applied using $\int_{\mathcal{T}_\ell} F(\mathbf{X}) d\mathbf{X} = \int_{\mathcal{T}_0} F(\psi^{-1}(\mathbf{x})) J^{-1} d\mathbf{x}$.

3. Consider the elliptic PDE (Poisson problem) given by

$$\Delta u(\mathbf{x}) = -f(\mathbf{x}), \quad \mathbf{x} \in \Omega, \quad u(\mathbf{x}) = 0, \quad \mathbf{x} \in \partial\Omega,$$

where $\Omega = [-L, L] \times [-L, L] \subset \mathbb{R}^2$. In the Ritz-Galerkin approximation, we seek a solution $u_h \in \mathcal{V}_h \subset \mathcal{V} = H_0^1(\Omega)$ with

$$a(u_h, w) = -\langle f, w \rangle_{L^2}, \quad \forall w \in \mathcal{V}_h,$$

where $a(u_h, w) = \int_{\Omega} \nabla_{\mathbf{x}} u_h(\mathbf{x}) \cdot \nabla_{\mathbf{x}} w(\mathbf{x}) d\mathbf{x}$ and $\langle f, w \rangle_{L^2} = \int_{\Omega} f(\mathbf{x}) w(\mathbf{x}) d\mathbf{x}$. Consider a basis of functions $\{\phi_k\}_{k=1}^N$ for \mathcal{V}_h . We can represent any $v \in \mathcal{V}_h$ by $v(\mathbf{x}) = \sum_i v_i \phi_i(\mathbf{x})$,

$u_h(\mathbf{x}) = \sum_i u_i \phi_i(\mathbf{x})$, and approximate f by $f_h(\mathbf{x}) = \sum_i f_i \phi_i(\mathbf{x})$. The FEM approximation u_h can be expressed as solving the linear system

$$\mathbf{A}\mathbf{u} = -\mathbf{M}\mathbf{f}.$$

The A is the *stiffness matrix* given by $A_{ij} = a(\phi_i, \phi_j)$, M is the *mass matrix* given by $M_{ij} = \langle \phi_i, \phi_j \rangle_{L^2}$, and $[\mathbf{u}]_i = u_i$, $[\mathbf{f}]_i = f_i$.

- (a) (Meshing) Discretize the domain Ω into elements $\mathcal{T} = \{\mathcal{T}_\ell\}_{\ell=1}^m$, where \mathcal{T}_ℓ are triangular elements. For the square domain $\Omega = [-L, L] \times [-L, L] \subset \mathbb{R}^2$, one way to discretize is to define a coarse mesh. A basic algorithm to obtain a more refined discretization is to loop over each triangle and bisect the edges to obtain four smaller triangles, see Figure 3. Data structures for this are a list of vertices $\mathbf{v}_i \in \mathbb{R}^2$ and tuples (i_1, i_2, i_3) which give the indices of the vertices of each triangle.

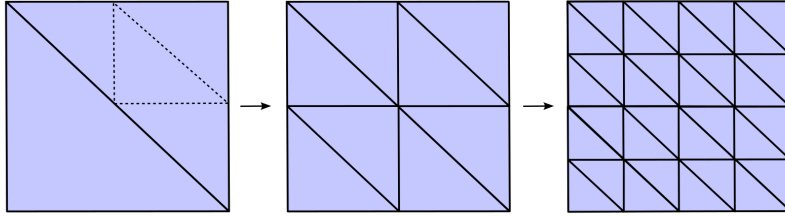


Figure 3: Mesh triangulation and refinement by triangle bisection.

Implement this meshing algorithm for the triangulation in Figure 3. Plot the triangulations when this refinement procedure is done up to $n = 5$ times.

- (b) (Assembly and Quadratures) For the discretization into triangular elements $\mathcal{T} = \{\mathcal{T}_\ell\}_{\ell=1}^m$, take $\{\phi_k\}_{k=1}^N$ to be the nodal basis functions for Lagrange elements with polynomial shape functions of degree d so that $v_h|_{\mathcal{T}_\ell} \in \mathcal{P}_d$. The stiffness matrix A is obtained through an assembly procedure where we compute the integral by breaking it into parts summing up the inner-products over each element \mathcal{T}_ℓ as $A_{ij} = a(\phi_i, \phi_j) = \sum_{\ell=1}^m \int_{\mathcal{T}_\ell} \nabla_{\mathbf{x}} \phi_i(\mathbf{x}) \cdot \nabla_{\mathbf{x}} \phi_j(\mathbf{x}) d\mathbf{x} = \sum_{\ell=1}^m A_{\ell,ij}$, and similarly, $M_{ij} = \langle \phi_i, \phi_j \rangle_{L^2} = \sum_{\ell=1}^m \int_{\mathcal{T}_\ell} \phi_i(\mathbf{x}) \phi_j(\mathbf{x}) d\mathbf{x} = \sum_{\ell=1}^m M_{\ell,ij}$. Integrals are approximated by high-precision quadratures

$$\tilde{A}_{\ell,ij} = \sum_k \omega_k \nabla_{\mathbf{x}} \phi_i(\mathbf{x}_k) \cdot \nabla_{\mathbf{x}} \phi_j(\mathbf{x}_k), \quad \tilde{M}_{\ell,ij} = \sum_k \omega_k \phi_i(\mathbf{x}_k) \phi_j(\mathbf{x}_k).$$

The $\{\omega_k\}$ are the quadrature weights and $\{\mathbf{x}_k\}$ are the quadrature nodes. Note in general the quadrature nodes can differ from the finite element nodes. We use these approximations to obtain

$$\tilde{\mathbf{A}}\mathbf{u} = -\tilde{\mathbf{M}}\mathbf{f}.$$

For the case of Lagrange elements using polynomial spaces of degree d , we use quadratures that have order $2d$. This allows for computing the integrals up to round-off errors. For quadratures on triangulations, see Figure 4 and Table 1.

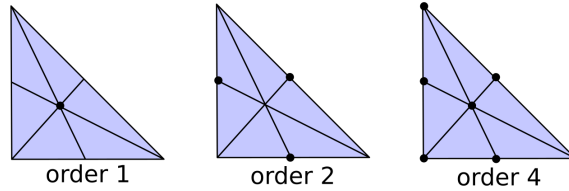


Figure 4: Quadrature Nodes.

Using this assembly + quadrature approach, implement codes to compute for a given triangulation the stiffness and mass matrices when $d = 1$ and $d = 2$.

Consider the FEM approximation for the solutions u with $L = \pi$ and (i) $u(x_1, x_2) = \cos(5x_1) \sin(5x_2)$ and (ii) $u(x_1, x_2) = \exp(-\cos(3x_1) + \sin(3x_2))$. Use $f(\mathbf{x}) = -\Delta u$ evaluated at the nodal points to obtain the numerical data for these test problems.

Make a log-log plot of the solution error vs mesh size $h^{-1} = 2^{-n}$ for meshes with refinements $n = 1, 2, \dots, 5$. What is the exhibited order of accuracy of the Lagrange FEMs when $d = 1$ and $d = 2$?

(c) (Iterative Methods) To solve approximately

$$A\mathbf{u} = \mathbf{b}, \text{ where } \mathbf{b} = -M\mathbf{f},$$

iterative methods can be used of the form

$$B\mathbf{u}^{n+1} = C\mathbf{u}^n + \mathbf{b}.$$

For convergence, $B - C = A$ and the spectral radius of $B^{-1}C$ is taken to satisfy $\rho(B^{-1}C) < 1$. It is common to decompose the matrix as $A = D - L - U$, where D is the diagonal entries, $-L$ the lower entries, and $-U$ the upper entries. A few example iterative methods are

- i. Direct Relaxation with $B = I$ and $C = I + \eta A$, with small enough η s.t. $\eta \leq 2/\lambda$ or smaller, where λ is the largest eigenvalue of A .
- ii. Jacobi Iteration with $B = D$ and $C = L + U$.
- iii. Gauss-Seidel Iteration with $B = D + L$ and $C = U$.

Compare these methods for approximating the solution \mathbf{u} when $L = \pi$ and (i) $u(x_1, x_2) = \cos(5x_1) \sin(5x_2)$ and (ii) $u(x_1, x_2) = \exp(-\cos(3x_1) + \sin(3x_2))$. Use $f(\mathbf{x}) = -\Delta u$ evaluated at the nodal points to obtain the numerical data for these test problems.

Make a log-log plot of the number iterations and the error for meshes with $n = 5$ refinements. How many iterations does each method need to converge to 1% accuracy for solving the linear system? We remark that in practice these convergence rates are further enhanced by using preconditioners.