Paul J. Atzberger

206D: Finite Element Methods University of California Santa Barbara

MATH 206D: Finite Element Methods

Welcome to MATH 206D: Finite Element Methods!

We will use the following books:

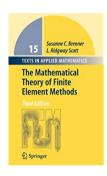
- Finite Elements: Theory, Fast Solvers, and Applications in Solid Mechanics (third edition), D. Braess.
- The Mathematical Theory of Finite Element Methods (third edition),
 - S. Brenner and R. Scott.

For more information, see the course website:

http://teaching.atzberger.org/

I look forward to working with you this quarter.





Introduction to Finite Element Methods

Variational Principle

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We find that

$$(\delta E[u])(v) = \int_0^1 u'(x)v'(x)dx + \int_0^1 v(x)f(x)dx$$

= $[u'(x)v(x)]_0^1 - \int_0^1 (u''(x) - f(x)) v(x)dx.$

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$$= \left[u'(x)v(x)\right]_0^1 - \int_0^1 \left(u''(x) - f(x)\right)v(x)dx.$$

Suggests "natural boundary conditions" $\rightarrow u'(0) = u'(1) = 0$.

Variational Principle
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Implies PDE holds (strong form)

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We take for now $\mathcal{V} = \{v \in L^2[0,1], a(v,v) < \infty, v(0) = 0\}$ (need to refine later).

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Proof Given that S is finite dimensional it has a basis $\{\phi_i\}_{i=1}^N$. Any function $u_S \in S$ can be expressed as

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We have "stiffness matrix" $[K]_{ij} = a(\phi_i, \phi_i)$ and "load vector" $[\mathbf{f}]_i = (f, \phi_i)$.

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Shows the problem has a solution.

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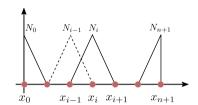
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Shows the problem has a solution.

Still, need theory to show $u_S \to u$ as $S \to V$ (i.e. we recover solution to the PDE in limit).

Consider space $\mathcal S$ generated by

$$v(x) = \sum_{i=1}^{n+1} v_i \phi_i(x)$$

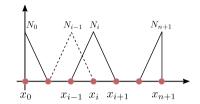


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(Hat Functions).

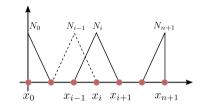
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Here, $h_i = x_{i+1} - x_i$ and $N_i(x_j) = \delta_{ij}$.



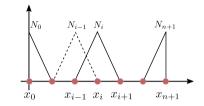
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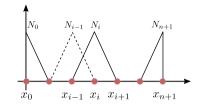
Mesh: $x_0, x_1, ..., x_{n+1}$. Elements: $e_i = \{x | x_{i-1} \le x \le x_{i+1}\}$. Shape Functions: $N_i(x)$.

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$$N_{i}(x) = \left\{ \begin{array}{ll} (x - x_{i-1})/h_{i-1}, & x \in [x_{i-1}, x_{i}] \\ (x_{i+1} - x)/h_{i}, & x \in [x_{i}, x_{i+1}] \\ 0, & \text{otherwise} \end{array} \right\}$$



(Hat Functions).

Here, $h_i = x_{i+1} - x_i$ and $N_i(x_j) = \delta_{ij}$.

Mesh: $x_0, x_1, ..., x_{n+1}$. Elements: $e_i = \{x | x_{i-1} \le x \le x_{i+1}\}$. Shape Functions: $N_i(x)$.

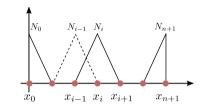
Let $S = \{v | v \in C[0, L], v(x) = \sum_{i=1}^{n} v_i N_i(x)\}$, referred to as the **shape space**.

Consider space $\mathcal S$ generated by

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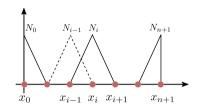
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Let $S = \{v | v \in C[0, L], v(x) = \sum_{i=1}^{n} v_i N_i(x)\}$, referred to as the **shape space**.

We would like to carry-out the Ritz-Galerkin approximations over this space.

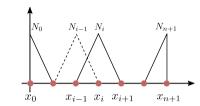
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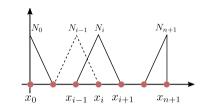


Consider the heat equation in 1D on [0, L]

$$\left\{ \begin{array}{ll} \frac{d^2u}{dx^2} = f(x), & x \in [0,L] \\ u(0) = T_1, u(L) = T_2, & x \text{ on boundary} \end{array} \right.$$

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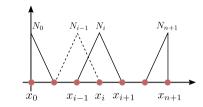
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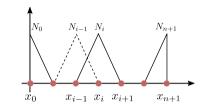
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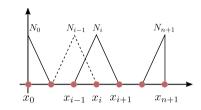
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To obtain stiffness matrix K and load vector \mathbf{f} , we need to compute the inner-products.

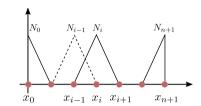
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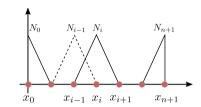


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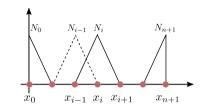
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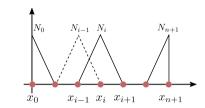
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$$a(u_{\mathcal{S}}, v) = (-f, v), \ \forall v \in \mathcal{S}, \qquad \mathcal{S} = \{v | v = \sum_{i=1}^{n} v_i N_i(x)\}$$

Stiffness matrix $K_{ij} = a(N_i, N_j)$ when $|i - j| \le 1$, $K_{ij} = 0$ otherwise.

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When $f = \sum_{i=0}^{n+1} f_i N_i(x)$, compute via "mass matrix" $M_{ij} = (N_i, N_j)$, and $[\mathbf{f}]_i = M_{ij} f_j$.

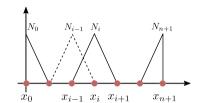
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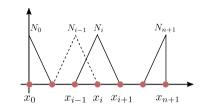
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Stiffness matrix when $h_i = h_0 = h$ and load vector when $f(x) = f_0$,

$$K = \frac{1}{h} \begin{bmatrix} 2 & -1 & 0 & \cdots & 0 \\ -1 & 2 & -1 & \cdots & 0 \\ 0 & -1 & 2 & \cdots & 0 \\ 0 & 0 & -1 & \ddots & -1 \\ 0 & 0 & 0 & \cdots & 2 \end{bmatrix},$$



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In this case, the $K\mathbf{u} = -\mathbf{f}$ has similarities to Finite Difference Method for the heat equation.

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$$a(u-u_S,w)=0, \forall w\in S$$

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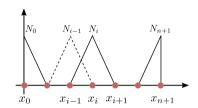
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Note,
$$\mathcal{S} \subset \mathcal{V}$$
.

$$||u-u_S||_E \leq \inf\{||u-v||_E|v\in S\}.$$

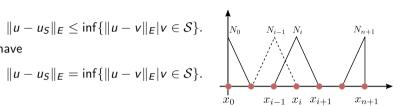


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Note, $S \subset V$. Since $u_S \in S$, we have

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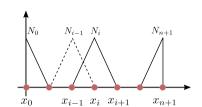
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The u_S is the best approximation possible when using energy norm to measure errors.



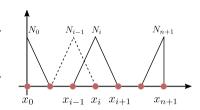
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Example (linear elements):

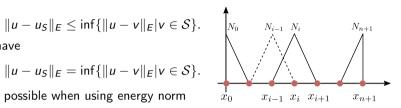
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Example (linear elements):

The Green's function for $-d^2u/dx^2 = f$ is given by

$$G(x,x_0) = \left\{ \begin{array}{ll} x, & x < x_0 \\ x_0, & x_0, \text{ otherwise} \end{array} \right\}, \quad \frac{dG}{dx} = \left\{ \begin{array}{ll} 1, & x < x_0 \\ 0, & x_0, \text{ otherwise} \end{array} \right\}, \quad \frac{d^2G}{dx^2} = -\delta(x-x_0).$$

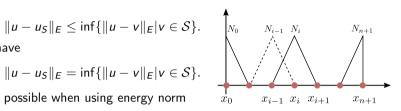
This gives

$$||u-u_S||_E \leq \inf\{||u-v||_E|v\in S\}.$$

Note, $S \subset V$. Since $u_S \in S$, we have

$$\|u - u_S\|_E = \inf\{\|u - v\|_E | v \in \mathcal{S}\}$$

The u_s is the best approximation possible when using energy norm to measure errors.



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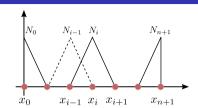
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The solution u above can be expressed as

$$u(x) = \int G(x, y) f(y) dy.$$

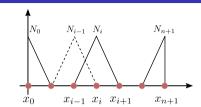
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The Green's function also has the property that

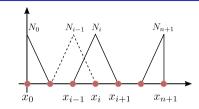
$$v(x_0) = a(v, G(\cdot, x_0)), \ \forall v \in \mathcal{V}$$



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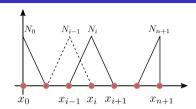
Putting this together we have the error can be expressed as

$$(u - u_S)(x_0) = a(u - u_S, G(\cdot, x_0)) = a(u - u_S, G(\cdot, x_0) - v), \quad \forall v \in S$$

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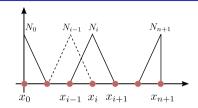
Since $G \in \mathcal{S}$ we have at the nodes x_1, x_2, \dots, x_n that

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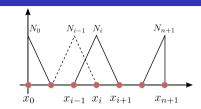
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This means u_S is piece-wise linear with $u_S(x_i) = u(x_i)$. We denote $u_S = u_I$ where u_I is the linear interpolation of the solution.

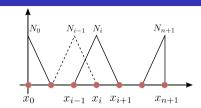
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Proof: We have $(u - u_I)(0) = 0 = (u - u_I)(h)$ since these are node locations.



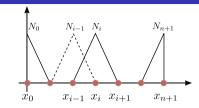
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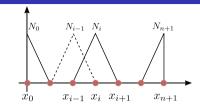
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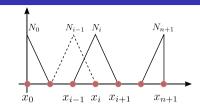


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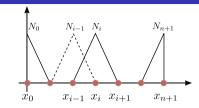
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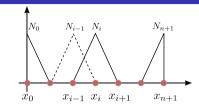
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$$\Rightarrow \|u - u_I\|_{\infty} \le Ch^2 \|u''\|_{\infty}. \blacksquare$$

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Key is to design function spaces and study their interpolation theory, since this indicates FEM errors.